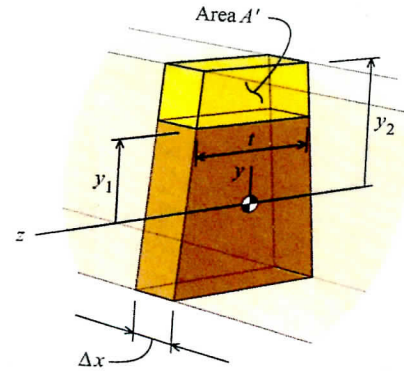
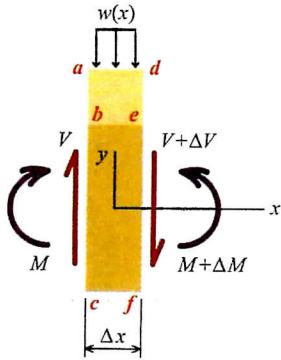


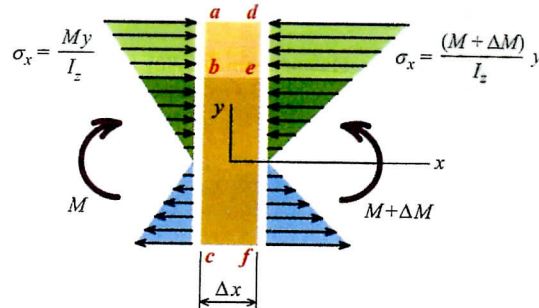
Prismatic beam subjected to non-uniform bending



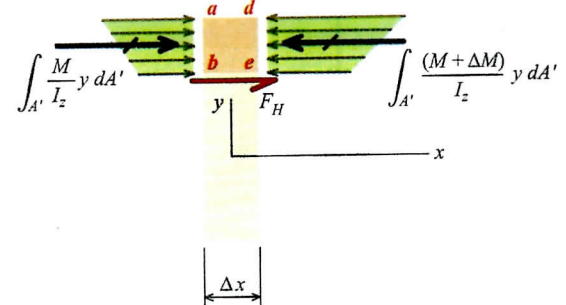
A slice of the beam with width Δx



Free-body-diagram of slice



Internal Bending Stresses



Free-body-diagram of area A'

In the last free-body-diagram set $\sum F_x = 0$ or : $F_{ab} - F_{de} + F_H = 0$ where:

$$F_{ab} = \int_{A'} \sigma_{ab} dA' \quad F_{de} = \int_{A'} \sigma_{de} dA' \quad \text{and} \quad F_H = \text{Shear force over the area } t \text{ by } \Delta x$$

Therefore:
$$\sum F_x = \int_{A'} \frac{M}{I_z} y dA' - \int_{A'} \frac{(M + \Delta M)}{I_z} y dA' + F_H = 0$$

Canceling terms and rearranging we get :
$$F_H = \int_{A'} \frac{\Delta M}{I_z} y dA'$$

Since with respect to area A', ΔM and I are constant, then :
$$F_H = \frac{\Delta M}{I_z} \int_{A'} y dA'$$

Note that the $\int_{A'} y dA'$ is the first moment of the area A' with respect to z-axis or Q, therefore:
$$F_H = \frac{\Delta M Q}{I_z}$$

Since $\tau_{ave} = \text{Force/Area}$, then
$$\tau_{H,avg} = \frac{F_H}{t \Delta x} = \frac{\Delta M Q}{t \Delta x I_z} = \frac{\Delta M Q}{\Delta x I_z t}$$
 Letting $\Delta x \rightarrow 0$ then $\Delta M / \Delta x$ becomes dM/dx

However $dM/dx = V$ or shear force, then the horizontal shear stress τ is :
$$\tau_H = \frac{V Q}{I_z t}$$

Since $\tau_{xy} = \tau_{yx}$, therefore $\tau_H = \tau_V = \tau$ or:
$$\tau = \frac{V Q}{I_z t}$$
 or

$$\tau = V Q / I_z t$$

And if we define the amount of the shear force over one unit length of the beam as **q**, or **shear-flow**, then:

$$q = \tau \times t = V Q / I_z$$

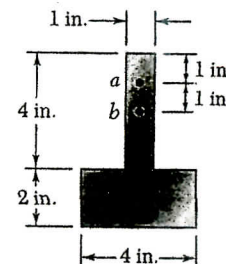
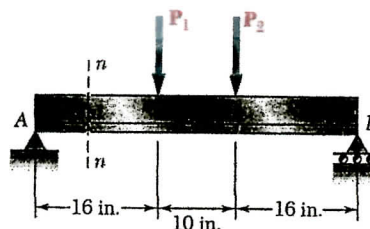
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Question 1:

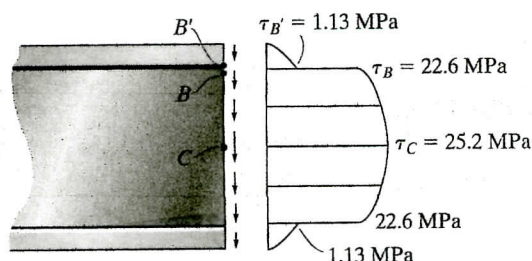
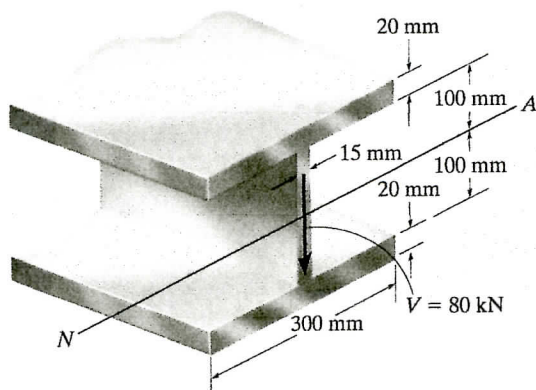
Given $P_1 = 8$ -kips and $P_2 = 18.5$ -kips, in the section n-n of the beam with cross section shown:

- a- Find shear stresses at points a and b.
 b- Find maximum shear stress in the section.
 c- Plot the shear stress distribution over this section.



Question 2:

A steel wide-flange beam has the cross section shown. If it is subjected to a shear force $V = 80$ -kN, plot the shear stress distribution over the beam's cross sectional area.

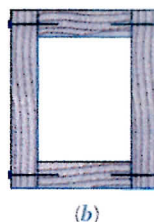
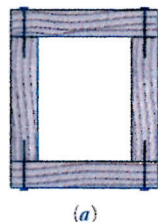


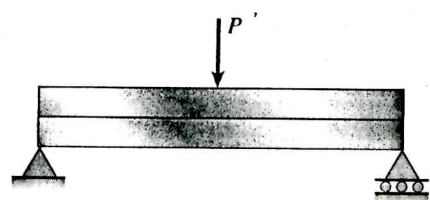
Result

Question 3:

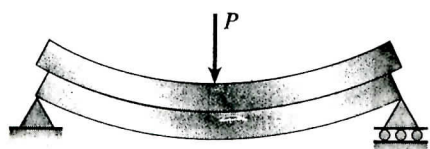
Consider a box beam obtained by nailing four pieces of wood together in two different arrangements.

The height of the beam is 200-mm, the width 120-mm and the thickness of each wood 20-mm. If each section is subjected to a shear force of $V = 3.5$ -kN and the spacing of the nails is 70-mm, determine the shear force in each nail.



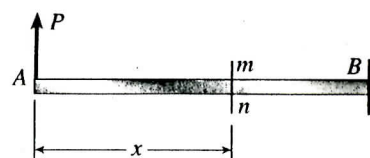


(a)

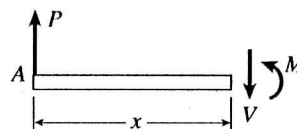


(b)

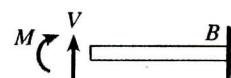
FIG. 5-27 Bending of two separate beams



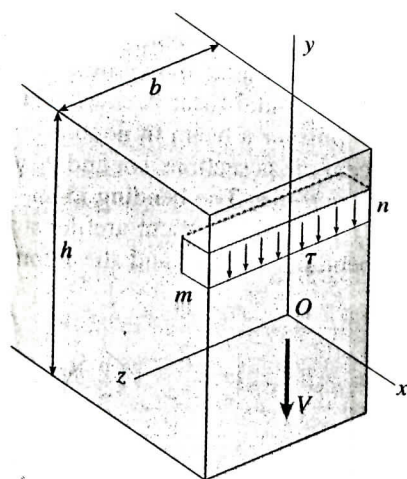
(a)



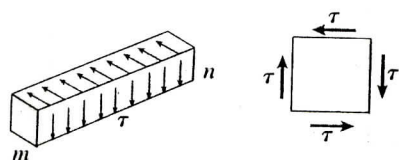
(b)



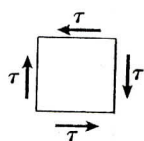
(c)



(a)

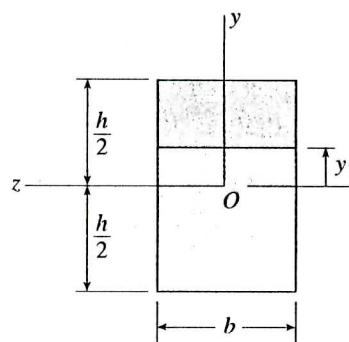


(b)

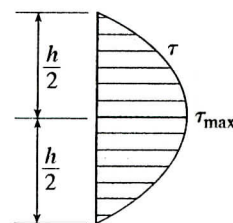


(c)

FIG. 5-26 Shear stresses in a beam of rectangular cross section

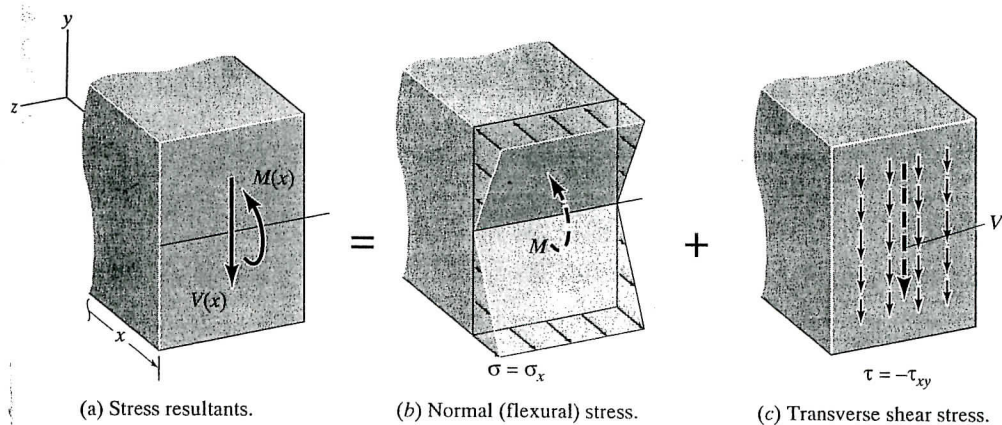


(a)

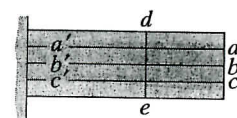


(b)

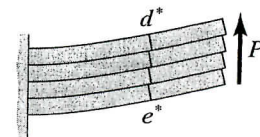
FIG. 5-30 Distribution of shear stresses in a beam of rectangular cross section: (a) cross section of beam, and (b) diagram showing the parabolic distribution of shear stresses over the height of the beam



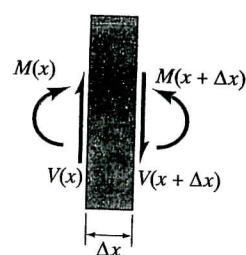
The stress resultants and stresses on a cross section.



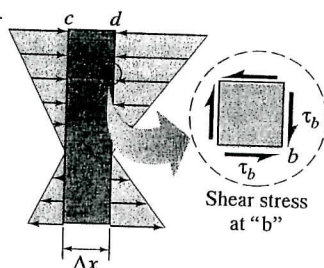
(a) A beam made of separate "planks."



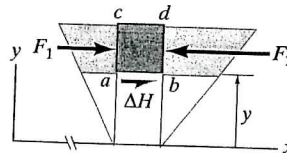
(b) Slip between non-bonded "planks."



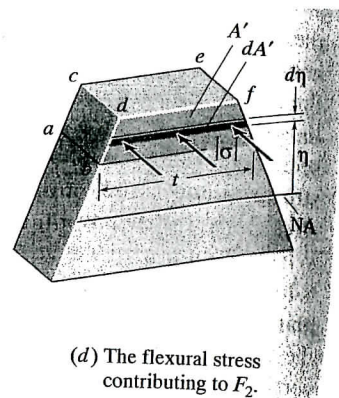
(a) An element of length Δx .



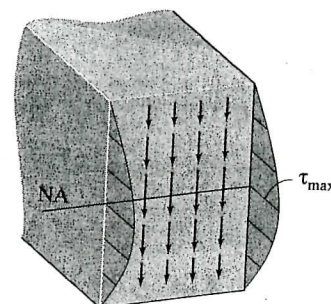
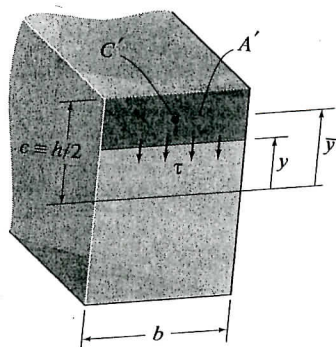
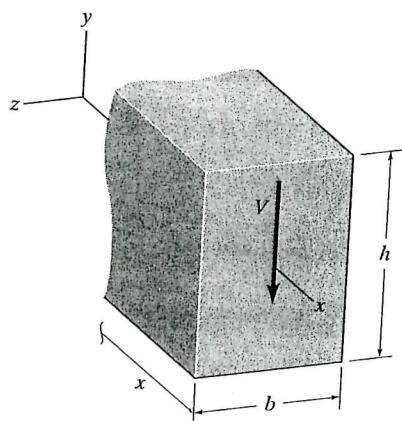
(b) The distribution of flexural stress.



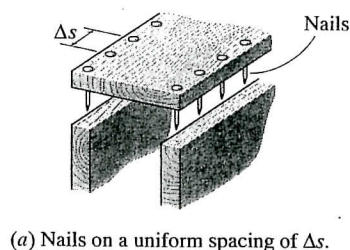
(c) A free body diagram (minus vertical shear on ac and bd).



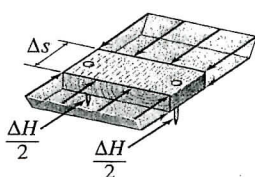
(d) The flexural stress contributing to F_2 .



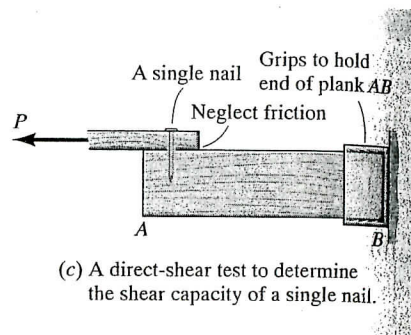
The shear-stress distribution on a rectangular cross section.



(a) Nails on a uniform spacing of Δs .



(b) A free-body diagram for determining shear force in discrete connectors.



(c) A direct-shear test to determine the shear capacity of a single nail.