

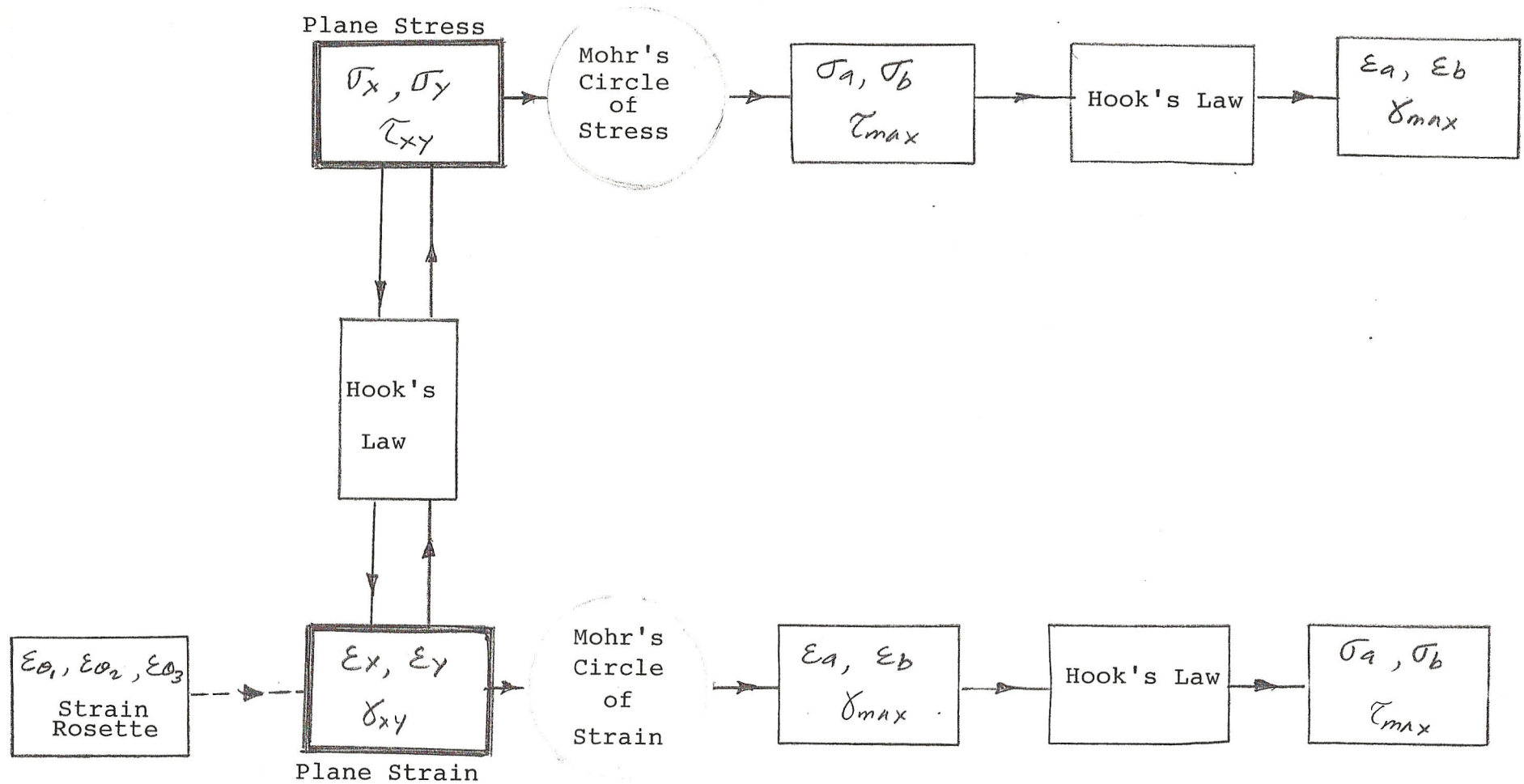
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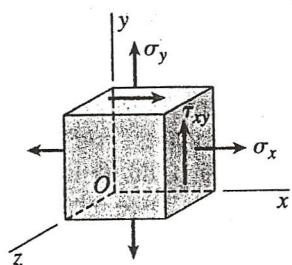
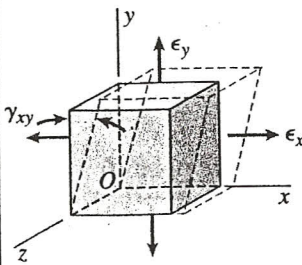
Background:

- 1- Strains due to Normal Stresses:
 $\epsilon_x, \epsilon_y, \epsilon_z$
- 2- Strains due to Shear Stresses:
 $\gamma_{xy}, \gamma_{yz}, \gamma_{xz}$
- 3- State of Stress: **Plane Stress**
- 4- Stress Transformation:
 - a- Purpose
 - b- Procedures: Using Equations or Mohr's Circle

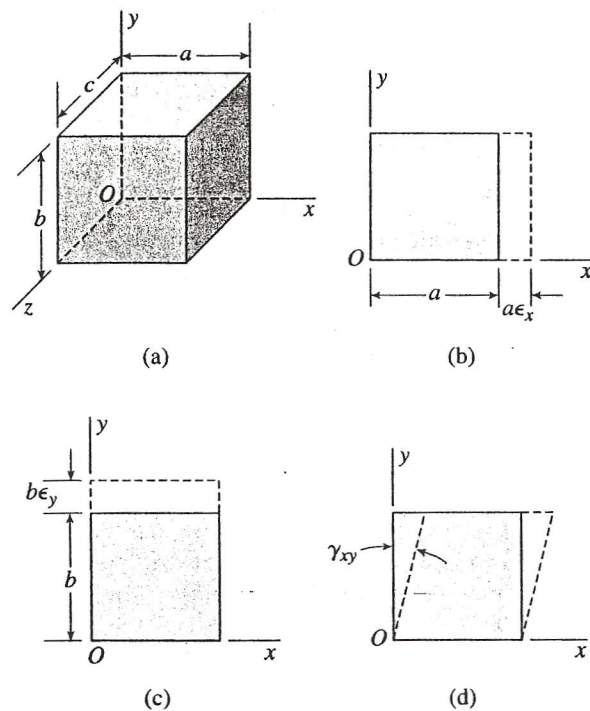
Today's Subject:

- 5- **Plane Strain**
- 6- **Strain Transformation:**
 - a- Purpose
 - b- Procedures: Using Equations or Mohr's Circle
- 7- Examples

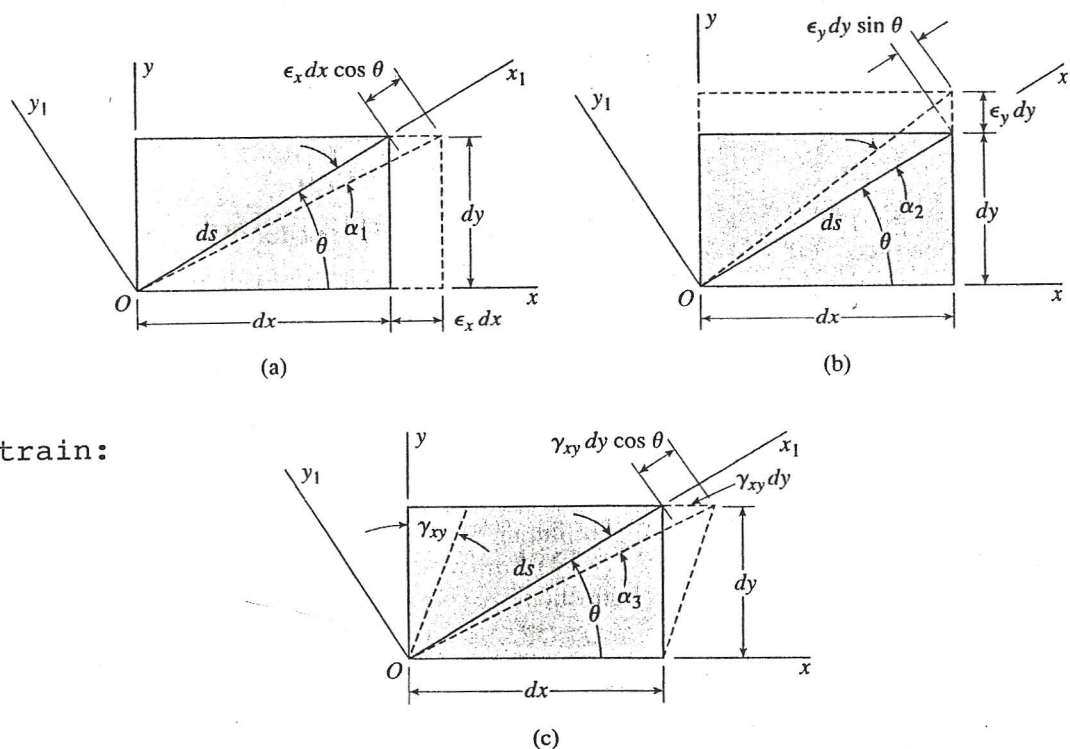


| | Plane stress | Plane strain |
|----------|---|---|
| |  |  |
| Stresses | $\sigma_z = 0$ $\tau_{xz} = 0$ $\tau_{yz} = 0$ σ_x , σ_y , and τ_{xy} may have nonzero values | $\tau_{xz} = 0$ $\tau_{yz} = 0$ σ_x , σ_y , σ_z , and τ_{xy} may have nonzero values |
| Strains | $\gamma_{xz} = 0$ $\gamma_{yz} = 0$ ϵ_x , ϵ_y , ϵ_z , and γ_{xy} may have nonzero values | $\epsilon_z = 0$ $\gamma_{xz} = 0$ $\gamma_{yz} = 0$ ϵ_x , ϵ_y , and γ_{xy} may have nonzero values |

Comparison of Plane Stress & Plane Strain



Strain Components ϵ_x , ϵ_y and γ_{xy} in xy plane



Deformation in Plane-Strain:

- a- Normal Strain ϵ_x
- b- Normal Strain ϵ_y
- c- Shear Strain γ_{xy}

Corresponding variables in
Transformation Equations of
Plane Stress vs. Plane Strain

Stresses

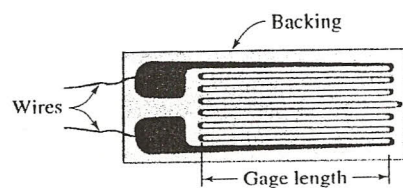
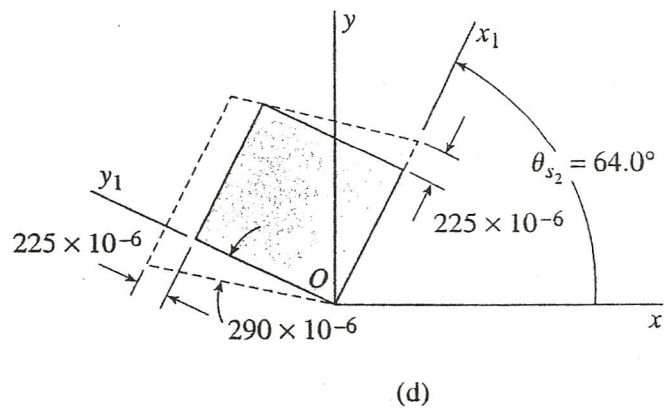
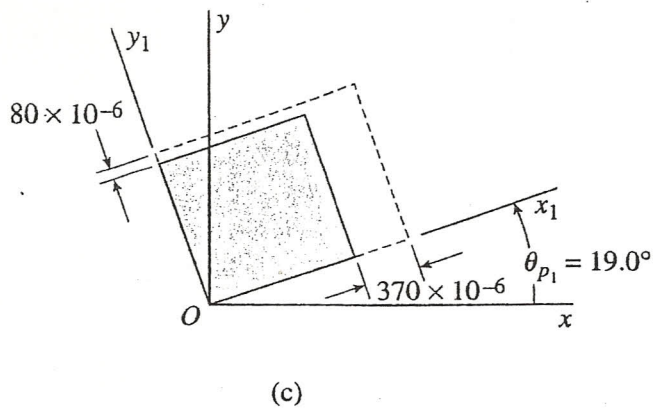
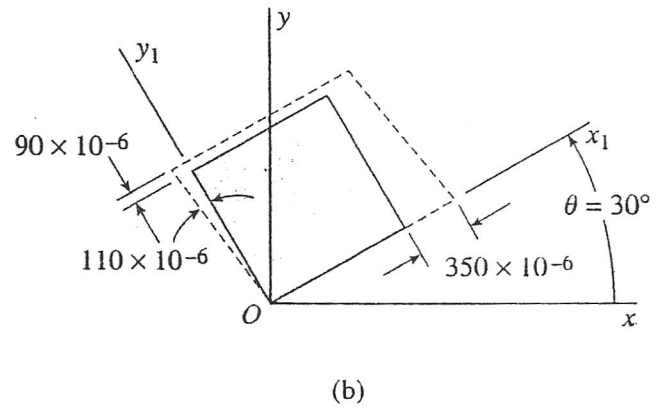
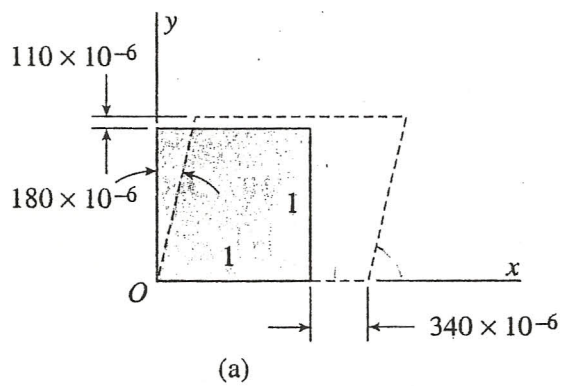
Strains

| | |
|------------------|----------------------|
| σ_x | ϵ_x |
| σ_y | ϵ_y |
| τ_{xy} | $\gamma_{xy}/2$ |
| σ_{x_1} | ϵ_{x_1} |
| $\tau_{x_1 y_1}$ | $\gamma_{x_1 y_1}/2$ |

Result of strain transformation of class exercise :

- a- Plane Strain Element
- b- Strain Element oriented @ $\theta = 30^\circ$
- c- Principal Strains Element
- d- Maximum Shear Strain Element

For details refer to Mohr's Circle.



Electrical-resistance strain gage (shown greatly enlarged).

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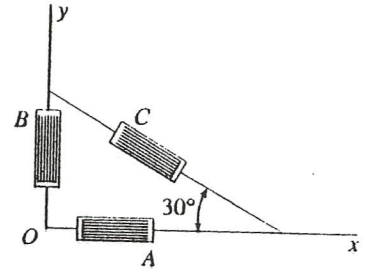
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Question 1: (20 Points)

On the surface of a space vehicle the strains are monitored by means of a strain rosette as shown. During certain maneuver the following readings were recorded: $\epsilon_A = 1,100 \times 10^{-6}$ $\epsilon_B = 200 \times 10^{-6}$ and $\epsilon_C = 200 \times 10^{-6}$.

The material is special magnesium alloy with: $E = 6,000\text{-ksi}$, $\nu = 0.35$, $G = 2,200\text{-ksi}$.

- a- Use **Mohr's Circle** to determine the principal strains and maximum shear strain γ at this point.
b- Find maximum shear stress τ and state whether it is in-plane or out-of-plane shear.



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Question 1:

The strains on the surface of an experimental device are monitored by means of three strain gages A, B and C put at $\theta = 60^\circ, 90^\circ$ and 120° respectively. The reading of the gages are $\epsilon_A = -90 \times 10^{-6}$, $\epsilon_B = -190 \times 10^{-6}$ and $\epsilon_C = -240 \times 10^{-6}$.

- Find the reading of a gage put at $\theta = 0^\circ$, along the x-axis.
- Use Mohr's Circle to determine the in-plane principal strains and their directions.
- Determine the absolute maximum shear strain and its plane.

Answers:

$$\epsilon_A = -90 \times 10^{-6}$$

$$\epsilon_a = -40 \times 10^{-6}, \quad \epsilon_b = -240 \times 10^{-6}, \quad \epsilon_c = 120 \times 10^{-6}$$

$$(\gamma_{\max})_{xz} = 360 \times 10^{-6}$$

Question 2:

A load $P = 72$ -kips is applied to the steel beam shown:

- Determine the reading of strain gages in 45° rosette at point A.
- Find principal strains and absolute maximum shear strain using strain transformation.

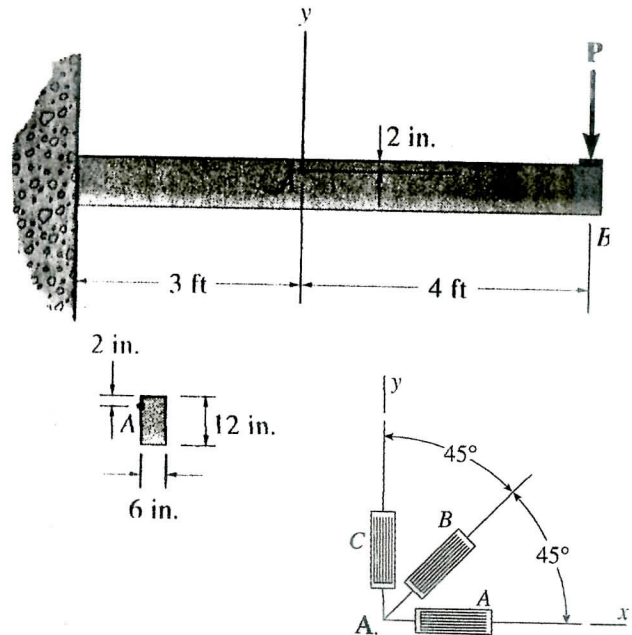
Assume: $E = 29,000$ -ksi $G = 11,500$ ksi

Answers:

$$\epsilon_x = 552 \times 10^{-6}, \quad \epsilon_y = -144 \times 10^{-6}, \quad \epsilon_{45} = 168 \times 10^{-6}$$

$$\epsilon_a = 554 \times 10^{-6}, \quad \epsilon_b = -146 \times 10^{-6}, \quad \epsilon_c = -144 \times 10^{-6}$$

$$(\gamma_{\max})_{xy} = 700 \times 10^{-6}$$



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Question 3

The spherical pressure vessel has an inner diameter of 2-m and a thickness of 10-mm. A strain gage having a length of 20-mm is attached to it, and it is observed to increase in length by 0.012-mm when the vessel is pressurized. Determine the pressure causing this deformation, the maximum in-plane shear stress, and the absolute maximum shear stress at a point on the outer surface of the vessel.

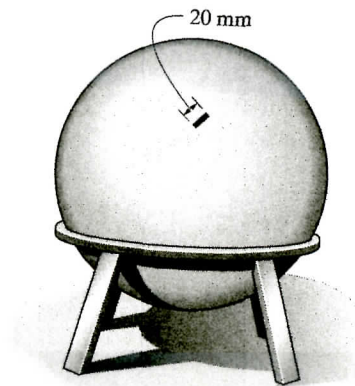
Assume: $E = 200\text{-GPa}$ $\nu = 0.3$

Answers:

$$p = 3.43 \text{ MPa}$$

$$\tau_{\text{max in plane}} = 0 \text{ MPa}$$

$$\tau_{\text{max out-of plane}} = 85.7 \text{ MPa}$$

**Question 4:**

A steel plate with $E = 30 \times 10^6 \text{ psi}$ and $\nu = 0.30$ is loaded in biaxial stress with $\sigma_x = 18,000\text{-psi}$ and σ_y as shown.

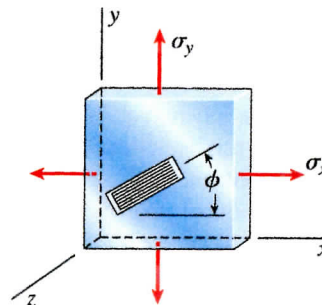
A strain gage put at $\phi = 30^\circ$ measures the strain $\epsilon_{30} = 407 \times 10^{-6}$, determine σ_y , the in-plane shear stress, the absolute maximum shear stress, in-plane shear strain and absolute maximum shear strain.

Answers:

$$\sigma_y = 2,400\text{-psi}$$

$$\tau_{\text{max in plane}} = 7,800\text{-psi}, \tau_{\text{max out-of plane}} = 9,000\text{-psi}$$

$$(\gamma_{\text{max}})_{xy} = 676 \times 10^{-6}, (\gamma_{\text{max}})_{xz} = 780 \times 10^{-6}$$



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Question 5

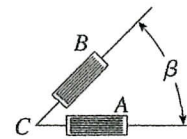
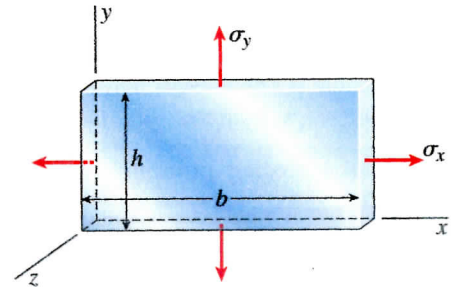
A thin rectangular plate in biaxial stress is subjected to σ_x and σ_y as shown. The plate has the width of $b = 10$ -in, the height of $h = 8$ -in, and thickness of $t = 1.5$ -in. After plate is loaded the strain gage on the surface of the plate show measurements of $\epsilon_A = 600$ - μ and $\epsilon_B = 448$ - μ at $\beta = 40^\circ$. Determine σ_x and σ_y .

Plate is made of brass with $E = 15,000$ -ksi and $\nu = 0.375$

Answers:

$$\sigma_x = 12,000\text{-psi}$$

$$\sigma_y = 8,000\text{-psi}$$

**Question 6**

A solid circular bar with a diameter of $d = 1.5$ -in is subjected to an axial force P and a torque T . After rod is loaded the strain gages mounted on the surface of the rod give readings of $\epsilon_A = 100$ - μ and $\epsilon_B = -55$ - μ . The rod is made of steel having $E = 30,000$ -ksi and $\nu = 0.29$. Determine:

a-The axial load P and the torque T .

b-The absolute maximum shear stress and shear strain in the rod.

Answers:

$$P = 5,300\text{-lbs}, T = 1,390\text{ lb-in}$$

$$\tau_{\max} \text{ in plane} = 2,580\text{-psi}, (\gamma_{\max})_{xy} = 222\text{-}\mu$$

